

## TESTING FLEXIBLE GRIPPERS FOR GEOMETRIC AND SURFACE GRASPING CONFORMITY IN RECONFIGURABLE ASSEMBLY SYSTEMS

C.I. Basson<sup>1\*\*</sup>, G. Bright<sup>1</sup> & A.J. Walker<sup>1</sup>

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#### Contact details

\* Corresponding author  
bassonc@ukzn.ac.za

#### Author affiliations

1 Discipline of Mechanical Engineering, University of KwaZulu-Natal, South Africa

# The author was enrolled for an MSc Eng (Mechanical) degree in the Discipline of Mechanical Engineering, University of KwaZulu-Natal, South Africa

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### ABSTRACT

The ongoing development in manufacturing technology facilitates flexibility in production and faces challenges of product handling. Flexibility was required in the form of adaptable grippers for robotic arms in pick-and-place procedures for reconfigurable assembly systems. A conceptual system was designed and tested according to the surface geometric conformity of grasped objects. The system proposed was a biologically inspired Fin Ray Effect<sup>®</sup> gripper. Grasping occurs due to the deformation of the rib structure of the appendage. The appendages were simulated for conformity by means of a finite element analysis, and performance was analysed by means of a physical sample mass test and a force test.

### OPSOMMING

Die deurlopende ontwikkeling in vervaardigingstegnologie fasiliteer buigsaamheid in produksie en word blootgestel aan uitdagings aan met betrekking tot produkhantering. Buigsaamheid word vereis in die vorm van aanpasbare robot gryp gereedskap vir robotarms in optel-en-plaas prosedures vir herkonfigureerbare monteringsstelsels. 'n Konseptuele stelsel is ontwerp en getoets aan die hand van die oppervlak meetkundige gelykvormigheid van gegrepe voorwerpe. Die voorgestelde stelsel was 'n biologies geïnspireerde Fin Ray Effect<sup>®</sup> gryper. Gryp vind plaas as gevolg van die vervorming van die ribstruktuur van die ledemaat. Die ledemaat is gesimuleer vir ooreenstemming deur middel van 'n eindige element analise, en verrigting is geanaliseer deur middel van 'n fisiese monster massa toets en 'n krag toets.

## 1 INTRODUCTION

High production rate capabilities are required in reconfigurable manufacturing systems, which are essential in the advancement of Industry 4.0. Reconfigurable assembly systems have been implemented in modern manufacturing systems to satisfy stakeholders' needs for functionality and high-capacity production response demands [1]. Pick-and-place procedures for assembly require the efficient grasping operation of a robotic arm and end-effector collaboration. A closed-loop robotic system integration was expressed as a conceptual model for a pick-and-place procedure. The system required should involve automation for higher repeatability, increased operational production, adaptive gripping characteristics, varied mass handling, and improved material handling [2]. The specifications should be met to improve the efficiency of the system.

A review of the literature presented a wide range of possible gripping techniques for flexible grasping methods. Agile gripping eliminates time consumption when interchanging fixtures for specific tasks. Research has shown that nature proved to be inventive about effective grasping, and corresponds with managing the challenges of material handling for reconfigurable assembly lines. Biologically inspired grippers have the potential to be incorporated into highly capable robotic systems. The flexible systems should aid the following recommendations for technological development and industrial actions: performance development and assembly measure, greater integration of

‘intelligent’ robotic fixturing, methodology development of human/robot collaboration, industrial gripper and sensor integration, human capability integration for autonomous robotic systems, path planning development for robotic arms, guidance of robotic hand through teaching processes, external support systems for robot system applications, combined perception systems and tactile sensing of grippers, and force control improvement for assembly systems [3].

Festo® has developed a number of biologically inspired grippers. The FlexShapeGripper® worked on the principle of a chameleon tongue [4]. Grasping by means of static surface pressure on the surface of the material, the FlexShapeGripper® proved to have excellent conformity. An alternative gripper designed by Festo® was the MultiChoiceGripper®, based on the conformity mechanisms of the Fin Ray Effect® [5]. A gripper system from Empire Robotics® called Versaball® incorporated the gripping action of granular jamming [6]. Robotiq® had also designed a gripper system based on the articulation of the human hand called the AdaptiveGripper® [7]. A conceptual design was created, based on the Fin Ray Effect®, with respect to the TIHRA® gripper [8] and the MultiChoiceGripper® illustrated in Figure 1. The exotic grippers described here have the potential to be implemented as flexible gripper systems to increase efficiency capabilities in production.

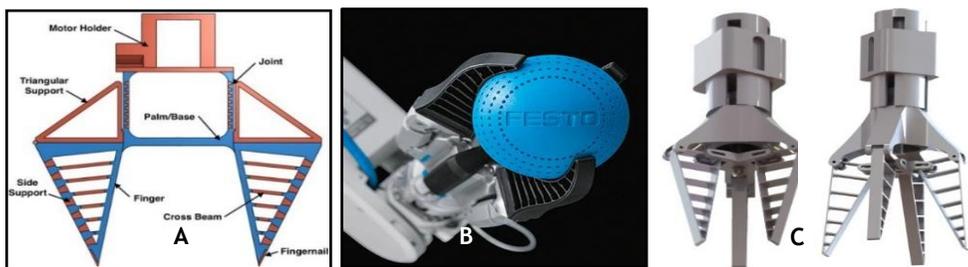


Figure 1: A) TIHRA gripper® [8]. B) MultiChoiceGripper® [5]. C) Proposed three-finger and four-finger grippers

## 2 SPECIFICATIONS FOR FLEXIBLE GRIPPING FROM THE LITERATURE

### 2.1 ISO specifications

The design gripper system should meet the requirements specified by the ISO 14539 standard, and be followed as closely as possible. The requirements presented by the ISO were effectively incomplete with regard to the specific procedure of testing. Quantitative requirements for gripper systems were not presented in the ISO specifications. The requirements were described as follows:

- The gripper to be developed should follow four grip states: A gripped state, a semi-gripped state, a laid state, and a free state.
- The gripping motion of the end-effector should consider the following conformity criteria: the degrees of freedom of the grasped object excluding friction forces, form closure grasping of 0-1 degrees of freedom excluding friction forces, form closure grasping of larger than 1 degree of freedom excluding consideration of friction force, external grasping of objects, and internal grasping of objects
- Forces have to be considered in terms of static and dynamic loading: Contact forces with respect to the contact point, contact line and contact plane with respect to the forces exerted from the fingers, and manipulating forces are considered to be the vector sum of all the exerted forces on the object.
- The following grasping stability criteria should be considered: The grasping stability of the initial state of the gripper pose should be restored after a disturbance force is applied, and the grasping stability of the contact grasp between the object and gripper should be restored after slippage.
- The intelligence of the gripper system in terms of feedback control should be considered: Object presence detection of verification and confirmation of successful gripping and releasing, finger positioning sensing in terms of overall object size and specified finger movement control, grasp force sensing relating to surface sensitivity of objects and higher stability grasps, external force sensing of the object weight, integrity of contact surfaces and object handling control,

and slip detections with regard to avoidance of handling of heavy objects and unstable gripping states.

- Specifications relating to the repeatability of robotic position control have been extracted from ISO 9283:1998 [9] and used as the basis for repeatability for gripping: The movement and object gripping of the objects should be 98 per cent repeatable, and the accuracy for repeatability should be measured with at least 10 measurements.

## 2.2 Dynamic force loading

Dynamic force loading was interpreted in terms of a force versus time grasp. The graph depicts force properties across a movement in the time domain of the manipulation. The force properties are detected in terms of the gripping performance and the criteria illustrated in Figure 2. The grip strength measurements were related to the grip fatigue of human hands over time, when grip force was measured [10]. A comparison of the grip strength of robotic grippers with human hands was established to serve as a comparable model.

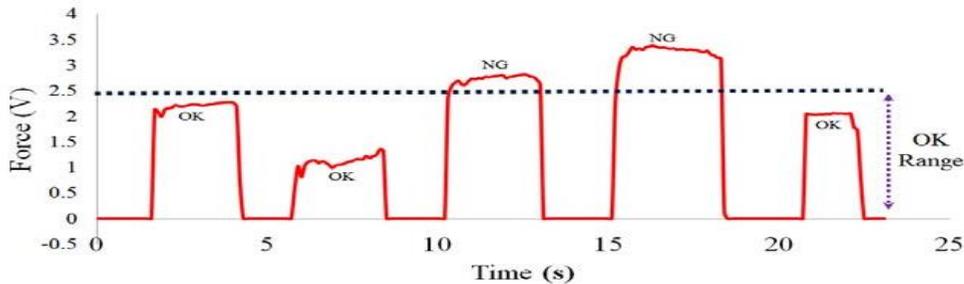


Figure 2: Gripping performance in terms of a force versus time graph [10]

## 3 CONCEPTUAL DESIGN

### 3.1 Design layout

The flexible gripping system was described as an integral arrangement that formed part of a closed-loop robotic system applicable to RMS [11]. The gripper system was planned according to the functionality of an end-effector framework, as shown in Figure 3. The system required an actuation input to manipulate the fingers. The gripper system followed a gripping procedure that encompassed the following: preparation for contact, establishing contact, fixturing and retention of the object, and releasing of the object at the correct position. The output of the mechanical system required efficient handling and correct grasping of the object. The grasping criteria were measured in terms of deflection of the appendages with regards to force handling capabilities of the fingers. The specifications for the gripper system procedure were generated by making use of ISO standard guidelines [12]. A typical gripping system consists of a robotic tool or end-effector, a control unit, and a robotic arm [13]. The mechanical system design of the flexible gripper system, illustrated in Figure 4, comprised a skeletal design, an actuation subsystem, robotic arm investigation, sensory system implementation, and a control system architecture.

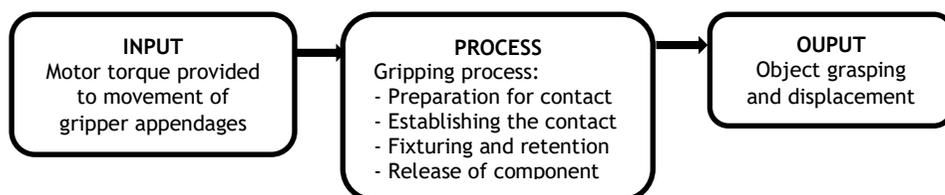


Figure 3: Gripper mechanical system procedure

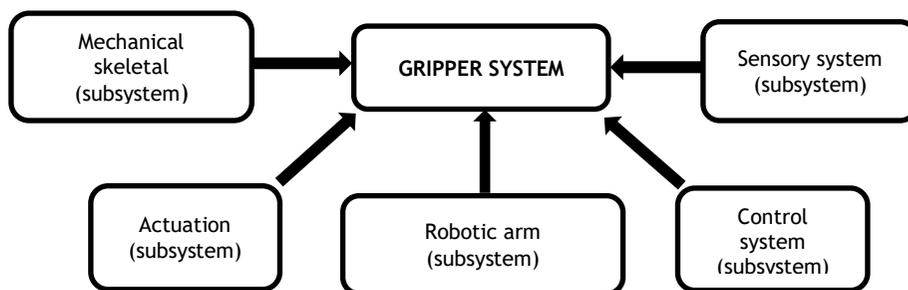


Figure 4: Gripper mechanical system outline

### 3.2 Fin Ray Effect®

The gripper concept selected was based on the conformity mechanism that was itself based on the deflection of fish fin ribs. The biologically inspired gripping occurrence was explained by Kniese of EvoLogics® who discovered the Fin Ray Effect® [14]. The appendage for the Fin Ray Effect® module consists of a V-shaped wall structure, containing a parallel rib structure in the shape of an A-frame. Conformity of the appendage occurs as a force is applied to the wall structure. The tip of the V-shape geometry tends to envelop the disturbance force, as illustrated in Figure 5. The unloaded structure is shown in A; the loaded structure is shown in B. The result of the shape adaption increases the surface friction contact with the object, which resembles the force. Additionally, the object is restricted from moving as a result.

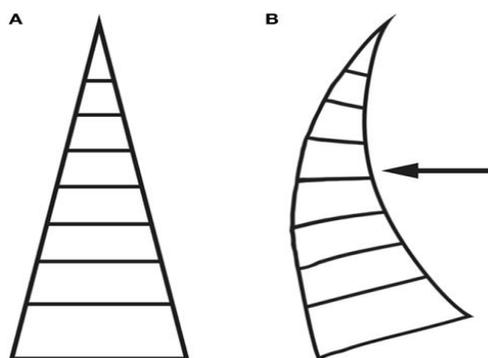


Figure 5: Mechanism of the Fin Ray Effect® [14]

## 4 SIMULATION FOR CONFORMITY

### 4.1 Materials selection

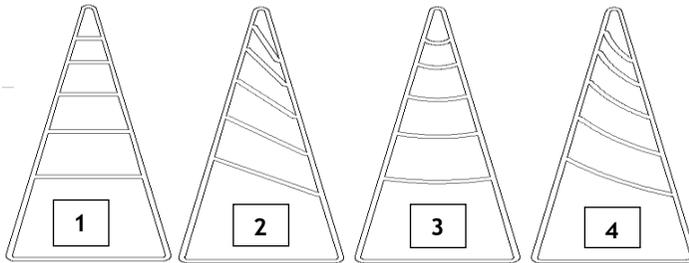
The materials were selected to enable 3D printing of components. The materials chosen for the simulation were acrylonitrile butadiene styrene (ABS) plastic and nylon. The reason for the choice of those materials was the high mechanical properties of printed components. The printed components were made from ABS; nevertheless, both nylon and ABS were simulated to compare the property attributes. The differences in grasping deflection and stress criteria were demonstrated in the comparison of the two selected materials. The material properties are described in Table 1. Care had to be taken when designing for 3D-printing material. The melting point had to be taken into account, the accuracy of the hole had to be considered, and the shrinkage factored had to be kept in mind [15].

Table 1: Material selection for appendage

Material	ABS plastic	Nylon
Elastic modulus [GPa]	2	1
Poisson's ratio [-]	0.4	0.3
Yield stress [MPa]	45	60
Print temperature [°C]	220-260	235-260

## 4.2 Rib structure configurations

The variation in the design of the rib structures of the gripper appendages consisted of geometric variations on the parallel Fin Ray Effect® rib configuration, as shown in Figure 6. The rib structures were designed according to the flexible and conformity properties required for effective shape grasping of objects. Geometry 1 illustrates the traditional rib configuration in the Fin Ray Effect® design. Geometry 2 shows an adapted version of the traditional structure, in which the ribs are slanted at specific angles. Geometry 3 possesses curved ribs, and has a parallel configuration similar to that of Geometry 1. Geometry 4 portrays comparable attributes related to Geometry 2 and Geometry 3, but the ribs are slanted and curved.



**Figure 6: Rib structure design for appendages**

The rib structure properties are described in Table 2. The rib units are placed at a 180° orientation from each other in the parallel alignment. Curved structures change accordingly as the V-shape of the wall structure changes to attain concentricity. Additionally, the slanted structures follow angled degree increments accordingly to the V-shaped wall angle change. The rib structures proposed were used as a conceptual rib structure design for the simulation in testing the conformity changes of the appendage in terms of deflection and stress criteria.

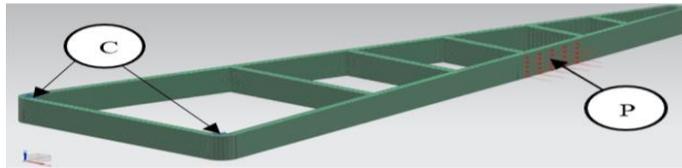
**Table 2: Rib orientation geometry**

Geometry	Rib orientation
1	Parallel (180°)
2	Slanted from the bottom (49.3°, 41.6°, 32.5°, 21.6°, 11.4°)
3	Curved from the bottom (r58.48[mm], r33[mm], r17.6[mm], r8.2[mm], r3.7[mm])
4	Curved from bottom (r77.3[mm], r55.9[mm], r39.6[mm], r29.9[mm], r25.3[mm]) and slanted from bottom (60.6°, 54.8°, 48.3°, 39.8°, 31.21°)

## 4.3 Simulation preparation

The simulation was executed using the Siemens NX® software package. A finite element analysis was performed to determine the deflection of the appendage structure. The simulation enabled the conception of a potential rib structure for effective conformity in grippers. As a product of the analysis, the result was used and rib geometry shapes were selected for the gripper design.

The appendage structure was designed on SolidWorks®, a computer-aided drawing (CAD) software package. The Parasolid models were imported into the simulation package illustrated in Figure 7. A force of 10 N was applied to the contact surface of the appendage wall in the form of a distributed load. The magnitude of the load P presents the minimum designed holding capacity of the gripper as 1 kg. The contact area created was 15 mm in length and 25 mm from the end point of the V-shape of the structure. The mesh of the model was made to be 1 mm grid size. The material was selected with nylon and ABS plastic properties for both simulation procedures. A fixed boundary condition was applied to area C, which resembles the hinged connection on the gripper assembly unit.

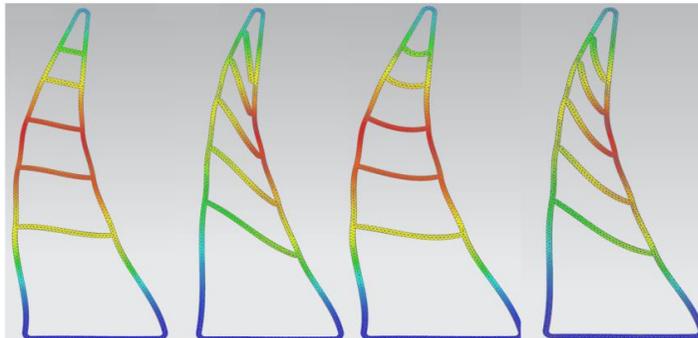


**Figure 7: Constraints and loads applied to Geometry 1**

A number of considerations were kept in mind for the simulation and design of the rib fingers. The post-simulation procedure required simulation convergence. The simulation's built-in software package produced an equation convergence graph, and verified that all calculated values and variables had been executed. The rib thickness adjoining the wall structure had to be kept in mind, as the deflection results would be affected. The attachment of the ribs to the V-shaped wall should be connected and have the same thickness at the joining surface for all ribs and the geometric structure. The geometry was manipulated by using fillet features in the CAD package.

#### 4.4 Simulation results

The displacement of the designed appendages simulation results is represented in Figure 8. Geometry 1 was treated as a reference for conformity because Geometry 1 represented the traditional Fin Ray<sup>®</sup> construct. The deflection in Geometry 2 increased by 14 percent compared with the reference geometry as a result of the slanted rib structure. Geometry 3 showed a slight increase of one per cent in the deflection criteria. Geometry 4 had an 11 per cent increase in deflection from the reference model. The simulation models showed that the manipulated rib structures had effects on the deflection and conformity properties of the appendage.



**Figure 8: Displacements of geometry 1, 2, 3 and 4, from left to right (see online version for colour)**

The simulation results for ABS plastic and nylon are described in Table 3 and Table 4 respectively. The stress distribution for both materials was constant due to the fact that the specific material selected had a minimal effect on stress, and the same force was applied throughout. Changes in deflection of the appendages were observed.

**Table 3: Tabulated results for ABS plastic**

	Deflection	Stress	Deflection $\Delta\%$	Stress $\Delta\%$
1	2.027 mm	14.90 MPa	Reference	Reference
2	2.317 mm	19.27 MPa	14.3	29.3
3	2.052 mm	15.28 MPa	1.2	2.6
4	2.248 mm	19.68 MPa	10.9	32.1

**Table 4: Tabulated results for nylon**

	Deflection	Stress	Deflection $\Delta\%$	Stress $\Delta\%$
1	1.013 mm	14.90 MPa	Reference	Reference
2	1.159 mm	19.27 MPa	14.4	29.3
3	1.026 mm	15.28 MPa	1.3	2.6
4	1.124 mm	19.68 MPa	11.0	32.1

## 5 ELECTRONIC CONTROL

### 5.1 Pseudocode

The pseudocode code presented in Figure 9 described the object manipulation process of the designed gripping system. The pseudocode described the operation of the gripping control procedure and the grip states throughout the pick-and-place procedure. The pseudocode initially identified the location of the object through a known position data input. The robot then identified the position of grasping. The grip contact and intensity was derived from a tactile force feedback loop. The micro-controller calculated and communicated the correct grip force required in the gripping phase. Finally, the object was released at the identified end location. The loop repeated until all object assemblies and movements had been completed.

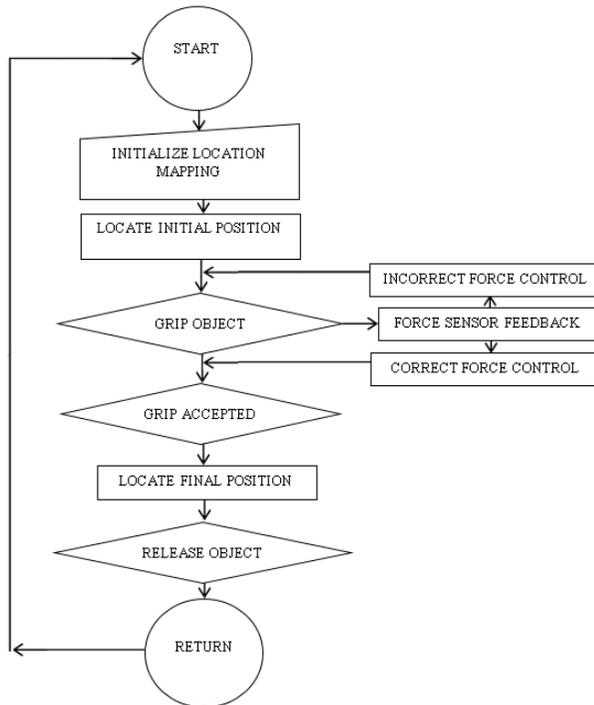


Figure 9: Pseudocode for gripper system

### 5.2 Force feedback control

The proposed sensory system regulated the force measured and the intensity of the force according to the desired force variable. The voltage signal received from force value conversion from the sensor was relayed to the microcontroller, as shown in Figure 10. The signal was transmitted to the gripper and measured against the desired variable. In the event that the new signal was not appropriate compared with the desired variable, a new signal was sent to the microcontroller.

The sensor position and force directions can be seen in Figure 11. The force signal relates to the magnitude of the force contact area between the object and the gripping surface. The measured force should be greater than the friction force between the contact surfaces to reduce the chance of slipping. The force should also not be greater than the force required so as not to compromise the integrity of the object surface condition [16].

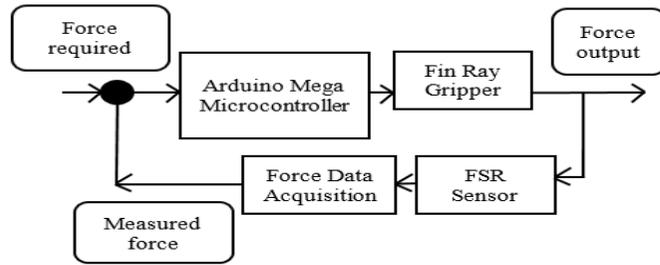


Figure 10: Force sensor feedback loop

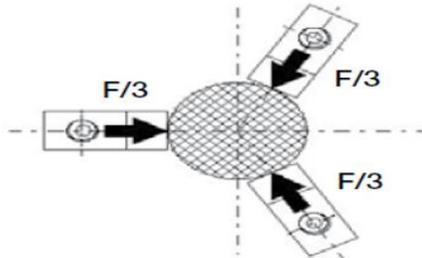


Figure 11: Force and sensor orientation in grip direction [16]

## 6 KINEMATIC AND DYNAMIC MODEL DESCRIBING FORCES ACTING ON MODEL

### 6.1 The requirement for kinematic and dynamic control of gripper system

The mathematical model focused on the flexible gripper and robotic arm sub-systems. The sub-systems consisted of a FANUC M10-iA industrial robotic arm and a three-finger and/or a four-finger Fin Ray® gripper. The design incorporates a force feedback system collaborating between the motor actuation of the end-effector and the electronic architecture. The force control criteria can be described theoretically from the output characteristic values of the mathematical model established for the robotic arm.

The dynamic force components manipulating the gripping properties were derived from the acceleration occurrence during a selected pick-and-place movement. The mathematical model was developed from the Denavit-Hartenberg model [17]. The potential performance criteria of the system were established in terms of the gripping force required by the design. The force output ranges were incorporated in selecting the actuation specification of the gripper. Additionally, a force feedback system should be incorporated to dynamically manipulate the load required from the grasping actuation. The mathematical model, therefore, expresses a potential dynamic model for force characteristics required for a closed robotic system for flexible gripper integration.

### 6.2 Denavit-Hartenberg model for FANUC M-10iA

The convention proposed was the Denavit-Hartenberg model, a widely used convention for the forward kinematics of robotic systems to determine the position of the end-effector. The DH-matrix condenses a six-by-six matrix that contains three rotational and three translational displacements into a four-by-four matrix containing two rotational and two translational displacements, shown in equation (1). The links of the serial robots are represented as the link length  $a_i$  and the link twist angle  $\alpha_i$ . The twist angle parameter designates the angle of the twist between  $i$  and  $i-1$ . The joints are additionally represented by the offset of the link  $d_i$  and the angular revolute of the joint  $\theta_i$ . The four parameters are used in the DH-matrix, shown in equation (2) .

$$A_i = [Rot_{z,\theta_i}][Trans_{x,d_i}][Trans_{x,a_i}][Rot_{x,\alpha_i}] \quad (1)$$

$$A_i = \begin{bmatrix} c\theta_i & -s\theta_i c\alpha_i & s\theta_i s\alpha_i & a_i c\theta_i \\ s\theta_i & c\theta_i c\alpha_i & -c\theta_i s\alpha_i & a_i s\theta_i \\ 0 & s\alpha_i & c\alpha_i & d_i \\ 0 & 0 & 0 & 1 \end{bmatrix} \quad (2)$$

The system to be described was the FANUC M-10ia robotic arm. The selected robotic arm consists of six degrees of freedom [18], therefore requiring six DH matrices, one for each joint. The degrees of freedom are shown in Figure 12. The DH model determines the operational dimensions and the work envelope of the specified robotic arm.

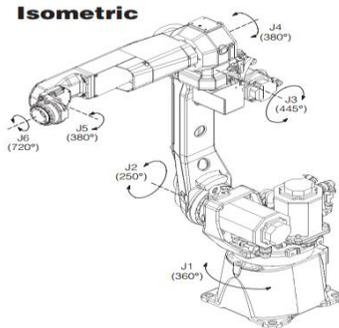


Figure 12: The joint orientation of the M-10iA [18]

The robotic arm can be represented in terms of the reference frames shown in Figure 13. The free body diagram describes the geometric dimensions and the elements for rotation in terms of the alignment of the joints [19]. The free body diagram enables the correct selection of the orientation of joints.

- 1) Z-axis is in the direction of the joint axis.
- 2) X-axis is perpendicular to both  $Z_n$  and  $Z_{n-1}$ .
- 3) Y-axis follows the right-hand rule.
- 4) The  $X_n$  axis must intersect the  $Z_{n-1}$  axis.

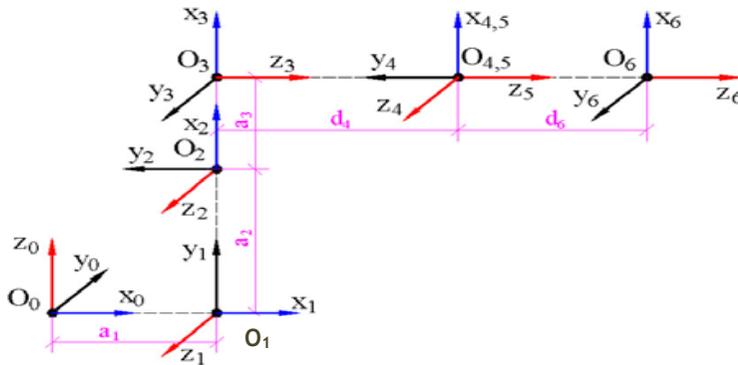


Figure 13: Coordinate reference frames for M-10iA robotic arm [19]

The joint arm links of the robotic arm are numbered 1 to 6, from the base to the end-effector of the arm. The origin reference frame from the base is denoted as  $O_0$  on a free body diagram. The position of joints 4, 5, and 6 affects the rotation of the tool axis and not the location of the tool [20]. The offset of joint 4 is shifted to joint 5, due to the singularities it creates on the DH-matrix entries. The input geometries are shown in Table 5.

Each joint has a corresponding homogeneous transformation matrix. The transformation matrices are described in equation (3) for each of the joint sets. The sets of matrices are multiplied by one another to describe the position matrix of the end-effector in equation (4). The homogeneous matrix  $T_n^0$  describes the displacements in the translation plane of the x, y, and z axes. The translation

displacements are expressed as  $P_x$ ,  $P_y$ , and  $P_z$  as described by equation (5). The position magnitudes are generated to calculate the acceleration properties of the system.

**Table 5: Geometric inputs for the DH model**

$T_i$	$\alpha_i$	$a_i$	$\theta_i$	$d_i$
1	$\frac{\pi}{2}$	$a_1$	$\theta_1, \frac{\pi}{2}$	0
2	0	$a_2$	$\theta_2$	0
3	$\frac{\pi}{2}$	$a_3$	$\theta_3$	0
4	$-\frac{\pi}{2}$	0	$\theta_4$	$d_4$
5	$\frac{\pi}{2}$	0	$\theta_5$	0
6	0	0	$\theta_6$	$d_6$

$$\begin{aligned}
 T_1 &= \begin{bmatrix} c_1 & 0 & s_1 & a_1 \cdot c_1 \\ s_1 & 0 & -c_1 & a_1 \cdot s_1 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} & T_2 &= \begin{bmatrix} c_2 & -s_2 & 0 & a_2 \cdot c_2 \\ s_2 & c_2 & 0 & a_2 \cdot s_2 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} & T_3 &= \begin{bmatrix} c_3 & 0 & s_3 & a_3 \cdot c_3 \\ s_3 & 0 & -c_3 & a_3 \cdot s_3 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} \\
 T_4 &= \begin{bmatrix} c_4 & 0 & -s_4 & 0 \\ s_4 & 0 & c_4 & 0 \\ 0 & -1 & 0 & d_4 \\ 0 & 0 & 0 & 1 \end{bmatrix} & T_5 &= \begin{bmatrix} c_5 & 0 & s_5 & 0 \\ s_5 & 0 & -c_5 & 0 \\ 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 1 \end{bmatrix} & T_6 &= \begin{bmatrix} c_6 & -s_6 & 0 & 0 \\ s_6 & c_6 & 0 & 0 \\ 0 & 0 & 1 & d_6 \\ 0 & 0 & 0 & 1 \end{bmatrix}
 \end{aligned} \tag{3}$$

where:

- $a_1 = 150$  mm
- $a_2 = 600$  mm
- $a_3 = 200$  mm
- $d_4 = 640$  mm
- $d_6 = 370$  mm
- $c_i = \cos(\theta_i)$
- $s_i = \sin(\theta_i)$

$$T_n^0 = T \cdot T_2^1 \cdot T_3^2 \cdot T_4^3 \cdot T_5^4 \cdot T_6^5 \tag{4}$$

$$T_n^0 = \begin{bmatrix} s_x & n_x & a_x & P_x \\ s_y & n_y & a_y & P_y \\ s_z & n_z & a_z & P_z \\ 0 & 0 & 0 & 1 \end{bmatrix} \tag{5}$$

### 6.3 Lagrangian force model

The values generated by equation (5) are introduced into equation (6). Equation (6) describes the combined Jacobian matrix for the acceleration components of an object in motion [21]. The equation is derived from differentiating the combined Jacobian vector matrix for displacement to generate a velocity Jacobian matrix. The velocity Jacobian matrix is differentiated to generate an acceleration Jacobian matrix.

$$\ddot{X} = J(q)\dot{q} + \left[ \frac{d}{dt} j(q) \right] \dot{q} \tag{6}$$

where:

- $\ddot{X}$ : Acceleration vector matrix.
- $J$ : Displacement Jacobean matrix.
- $j$ : Velocity Jacobean matrix.
- $q$ : Displacement vector.
- $\dot{q}$ : Velocity vector.
- $\ddot{q}$ : Acceleration vector.

The applicable forces can be described by means of a Lagrangian torque model, describing the force components of a moving object, as shown in Figure 14. The mathematical model describes the inertia component A, the Coriolis component C, the centripetal component B, and the gravitational component G [22]. The acceleration and velocity vectors from equation (6) are used to compute the force vectors in equation (7).

$$A(q)[\ddot{q}] + B(q)[\dot{q}\dot{q}] + C(q)[\dot{q}] + G(q) = \tau \quad (7)$$

where:

$q$ : Vector of joint angles.

$A(q)$ : Symmetric, bounded, positive definite inertia matrix.

$C(q)$ : Coriolis forces.

$B(q)$ : Centripetal forces.

$G(q)$ : Gravitational force.

$\tau$ : Vector of actuator torques.

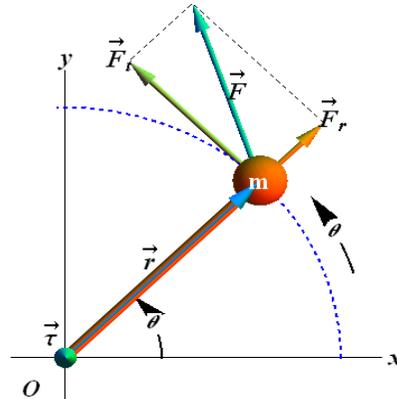


Figure 14: Force attributes of a moving object [22]

## 7 TESTING FOR GRIPPER CONFORMITY

### 7.1 Static holding mass test

A mass holding test was executed on the gripper system. The gripper was preloaded with 1630g, and increments of 100g were added until failure occurred due to slippage. The maximum loading mass range was 2435g as a selected criterion. The maximum mass holding range for geometry is shown in Figure 15 for the three-finger and four-finger grippers. The experiment was repeated 15 times to determine repeatability. The experiment was repeated for Geometries 1, 2, 3 and 4. Additionally, the experiment was performed on the three-finger and four-finger grippers. The repeatability of the gripping procedure resembles the probability of the gripper system being able perfectly to grip the maximum mass range. The slipping of the object during the experiment was due to a variety of factors, including a variation in grip force, slippage, eccentric grip, static coefficient irregularities, and surface characteristics.

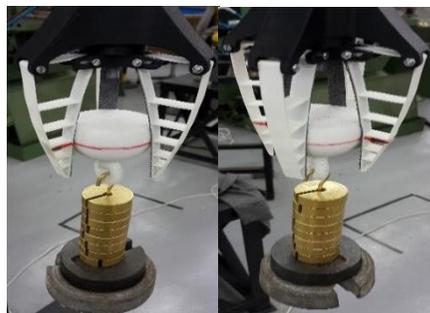


Figure 15: Mass holding test of three-finger and four-finger grippers

The results from the mass holding test for the three-finger and four-finger grippers are shown in Table 6 and Table 7 respectively. The average mass range and the repeatability rounded off performance for the three-finger gripper were 2.2-2.4 kg and 89-97 per cent respectively. The average mass range and the repeatability rounded off performance for the four-finger gripper were 2.3-2.5 kg and 83-99 per cent respectively. The ideal repeatability range would be 98 per cent; nonetheless, factoring in slippage conditions, a realistic repeatability would be 90 per cent.

**Table 6: Mass holding test for three-finger gripper**

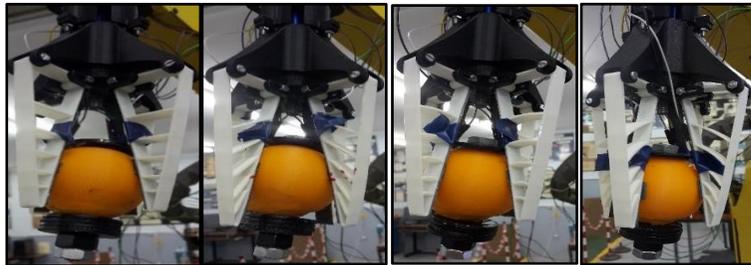
Geometry	Average mass	Repeatability
1	2177.05 g	89.2 %
2	2297.99 g	93.7 %
3	2338.34 g	92.4 %
4	2418.87 g	97.3 %

**Table 7: Mass holding test for four-finger gripper**

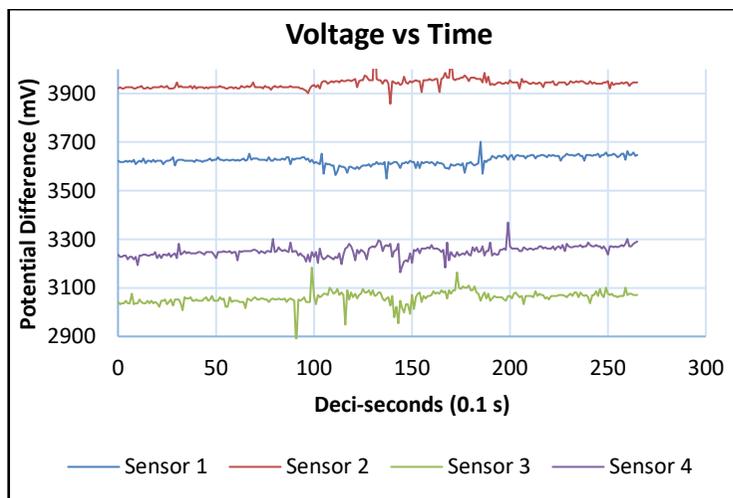
Geometry	Average mass	Repeatability
1	2344.80 g	93.4 %
2	2418.89 g	96.9 %
3	2398.64 g	95.9 %
4	2459.15 g	98.6 %

## 7.2 Dynamic holding force test

Dynamic testing was performed on the three-finger and four-finger gripper system. A spherical specimen weighing 320 g was grasped. The specimen needed to be concentric to enable direct contact with all the sensors. The testing was applied to Geometries 1, 2, 3 and 4, as shown in Figure 16. The specimen was grasped and manipulated according to a set path plan for the robotic arm. Voltage signal data was recorded according to time. The signal values were transmitted at a transferral rate of 10 Hz. The resulting data is shown in Figure 17 as a voltage versus time graph. The voltage signal represents the force applied to the object through the gripper fingers. The graph shows a steady signal, with the addition of a slight incline confirming self-conformity through dynamic movement. The experiment was repeated five times for accuracy. The dynamic performance of the three-finger and four-finger grippers is shown in Table 8 and Table 9 respectively.



**Figure 16: Dynamic test for four-finger gripper: Geometries 1, 2, 3 and 4**



**Figure 17: Voltage versus time graph for four-finger gripper: Geometry 1. (see online version for colour)**

**Table 8: Three-finger dynamic gripper performance**

Configuration	Dynamic loading (g)	Estimated grip force required (g)
Geometry 1	320	1660
Geometry 2	320	1607
Geometry 3	320	1687
Geometry 4	320	1257

**Table 9: Four-finger dynamic gripper performance**

Configuration	Dynamic loading (g)	Estimated grip force required (g)
Geometry 1	320	1504
Geometry 2	320	1420
Geometry 3	320	1479
Geometry 4	320	1146

### 7.3 Dynamic and static visual test for miscellaneous parts

Dynamic testing was performed to evaluate the gripping conformity through visual confirmation, as shown in Figure 18. Various mass and geometric specimens were gripped, and a dynamic movement representing a pick-and-place procedure was applied. The experiment was repeated five times each for the three-finger and four-finger grippers, incorporating Geometries 1, 2, 3 and 4. The results showed complete conformity and stable gripping throughout the experiment in all configurations.



**Figure 18: Visual dynamic and static testing of the four-finger gripper for Geometry 1.**

## 8 CONCLUSION

The biologically inspired gripper was analysed in terms of its conformity properties. The gripper appendages were designed according to the Fin Ray Effect®, and the ribs structure was altered accordingly for increased flexibility. The geometric rib structures were simulated for conformity in terms of displacement. The design considerations were taken into account, and a finite element analysis (FEA) proved that the altered rib constructs affected the conformity properties of the appendages in respect of deflection, as shown in Table 3 and Table 4. The operational force components for the object being gripped in the Fin Ray Effect® design were described using a Lagrangian model. The position values from modelling the robotic arm through a DH homogeneous transformation matrix were able to be used to determine the velocity and acceleration components for the Lagrangian model. A Jacobian matrix described the model to determine the acceleration components. The three-finger and four-finger gripper system, including the four geometries, were tested in a mass holding test. The static testing showed the increased repeatability and holding values of the modified rib structures, as shown in Table 6 and Table 7. The repeatability of the gripper system was attained through the experimentation process using the ideal and real repeatability ranges from the ISO specifications. Dynamic testing produced an better performance for the modified rib structures, as shown in Table 8 and Table 9.

The modified rib structures proved to have better gripping characteristics than those of the traditional Fin Ray Effect® (Geometry 1) rib configuration, as shown in Table 10. That qualitative comparison table outlines how the performance increased through the modification of gripper finger geometries. The most improved performance criterion was determined to be Geometry 4, which consisted of a circular and skewed geometric construct. High stress zones were avoided by including rounded structures and angled geometries to reduce high stress flow along the material.

**Table 10: Qualitative comparison of rib structure geometries for the three-finger and four-finger grippers**

Gripping geometry	Conformity deflection (FEA)	Repeatability	Maximum static loading	Estimated required grip force	Varied shape holding
1	Baseline	Baseline	Baseline	Baseline	Baseline
2	Highest improvement	Improved	Improved	Improved	Similar
3	Improved	Improved	Improved	Improved	Similar
4	Improved	Highest improvement	Highest improvement	Highest improvement	Similar

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